

A Rail-Road Hybrid Vehicle: *Dynamic Stability Analysis*

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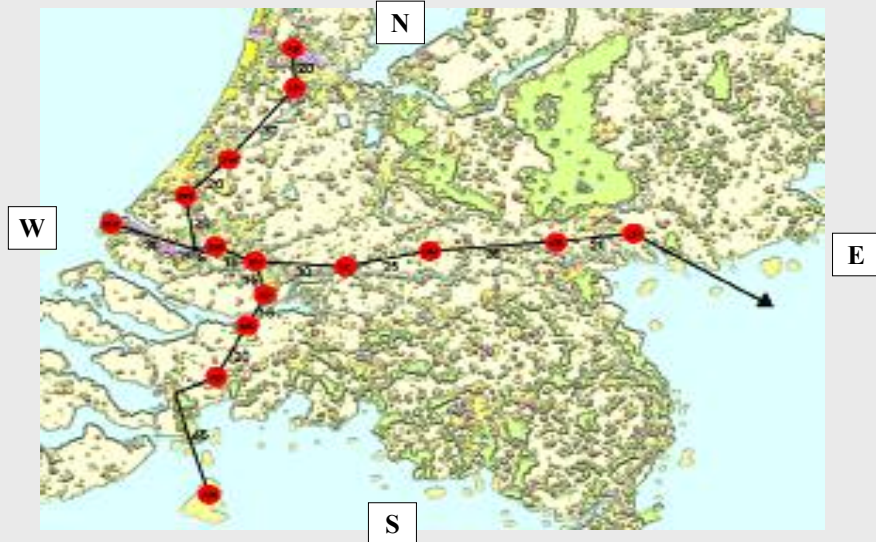
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Introduction: Rail-Road Hybrid Betuwe Route



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Introduction



Combining Rail & Road

- ❑ Rail vehicles:
 - + Low rolling resistance
 - - Switching / flexibility

- ❑ Road vehicles:
 - + Flexibility in manoeuvring
 - + Door-to-door transport
 - - High rolling resistance

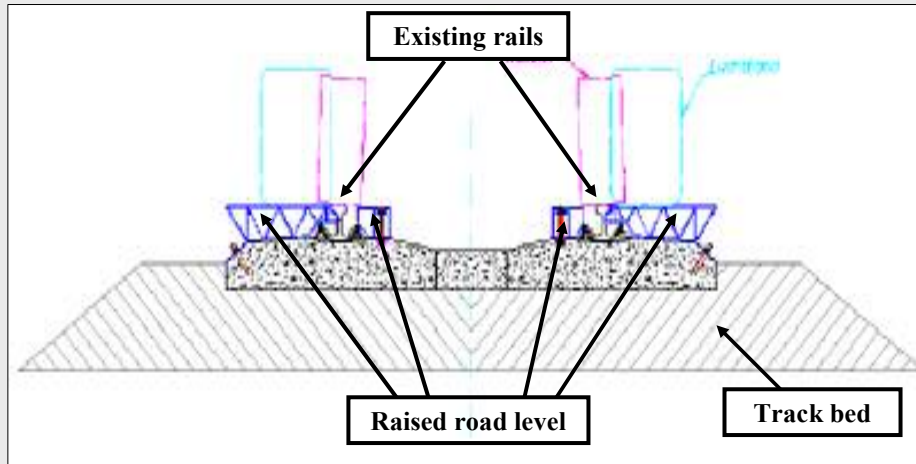
- ❑ → The rail-road Hybrid vehicle
 - Road vehicle with up to 60 % of vehicle load on simple and light rail axles.

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Introduction

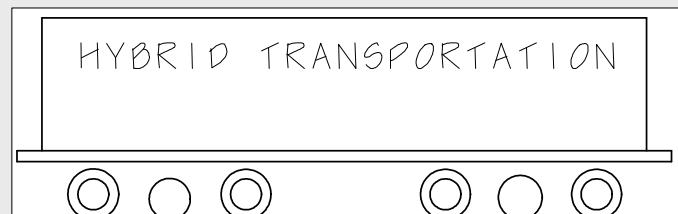


Proposed Rail-road hybrid track



Hybrid vehicle concept

- Rail-road hybrid vehicle:
 - intelligent autonomous road vehicle (agv)
 - Active steered & driven road wheels
 - electronically lateral guiding



- light rail axles: no brakes & self centring
- vertical force control on rail wheels.

The Hybrid: system layout

- Autonomous, on-board power source
- General air spring suspensions applied
- Total vehicle mass approx. 45 tonnes

- 4 road axles: 1 – 3 4 – 6
- 2 rail wheel sets: 2 5

- Point follower steering on axle 1 and 6
- Ackermann steer angles on axles 3 and 4.

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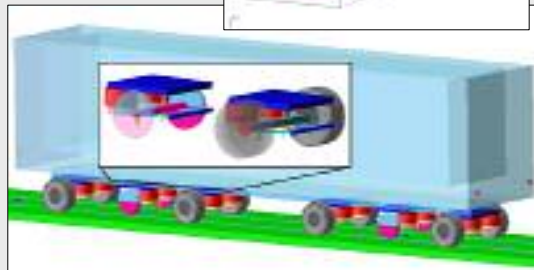
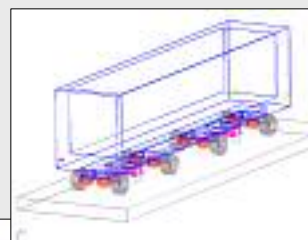
System layout



ADAMS model approach

- Fully parametrised model using UDE's
- 2 Axle groups, each 2 road & 1 rail axle
- Template based model components
- Generic axle models applied
- Pivot steered axles with torsion spring
- State-of-the art *Magic Formula* tires

- Rail axles *go-with-flow*
- Generalised rail contact
- Possible zero-friction & conicity in rail wheels
- Vehicle load from 0 to 100 % on rail axes.



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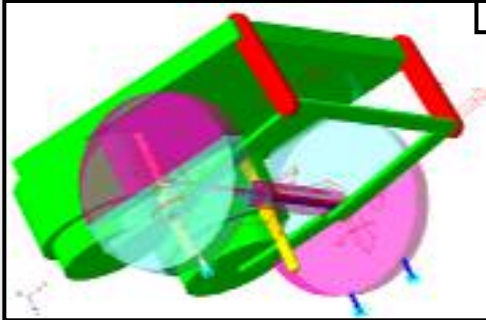
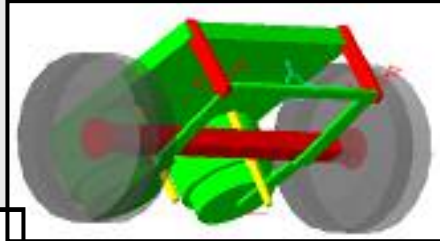
System layout



Axle suspension models

Rail axles

- Airspring with rigid link
- Axle lift (0 % rail load)
- Conical wheels – single point contact wheel-to-rails
- Possible 3-D rail curvature



Road axles

- Airspring with rigid link
- Axle lift (0 % road load)
- Range of MF tyre models
- Variable steering method
- Applied: Point follower with 2 m and 3 m lead point.

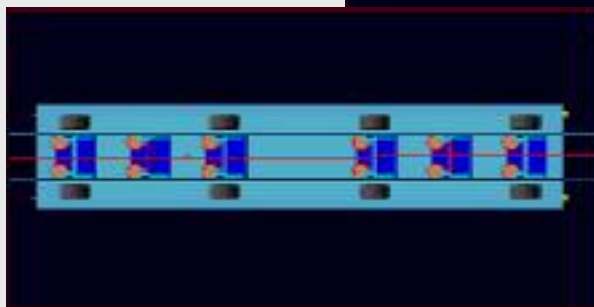
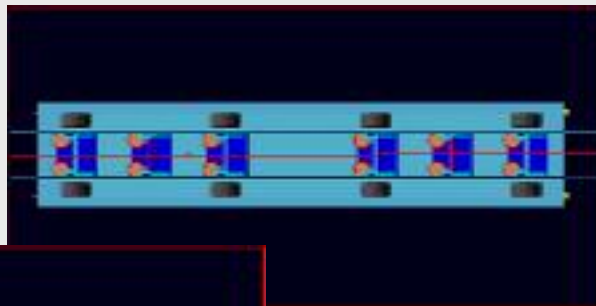
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System layout



Axle steering system

Axle 1 steer input →



← Axle 6 steer input

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System layout



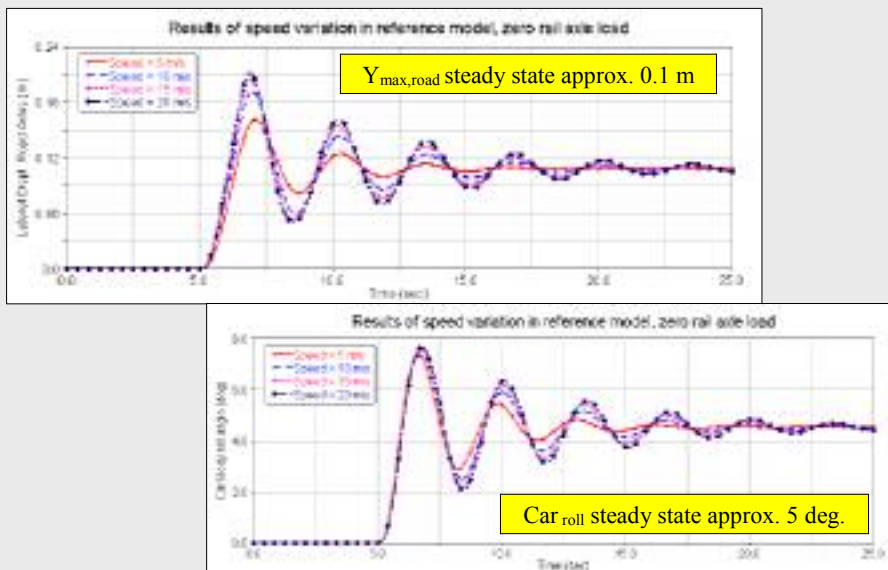
Simulation approach

- Analyse stability using lateral wind input:
 - Wind force amplitude equal to 0.1 G lat. acc.
 - Good system damping around speed of 55 km/h
 - Max. lateral deviation must be smaller than 0.10 m
- Assess vehicle and steering control parameters
- Model settings used:
 1. Completely on road wheels: → reference case
 2. Rail wheels: zero α & μ rails: → worst case (level icy rail)
 3. Steering system optimal parameters (PID required ?)
 4. Complete system verification on rails.

Simulation output

- Signals:
 - $Y_{\max,road}$: max lat. displ. of any road axles
 - $F_{z,rail,\%}$: sum of rail axle forces vertical, % of total
 - Car_{roll} : dynamic car body roll angle
- Design parameter studies: → tendencies
- Methods:
 - Peak value (overshoot)
 - Steady state (static response).

1. Reference car: simulation results

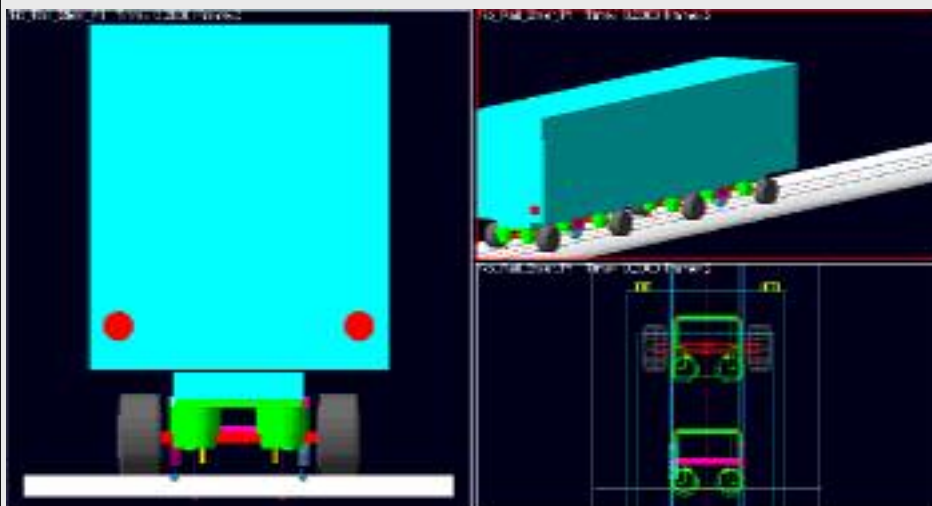


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Results: 1 Reference



Reference vehicle at zero vertical rail forces

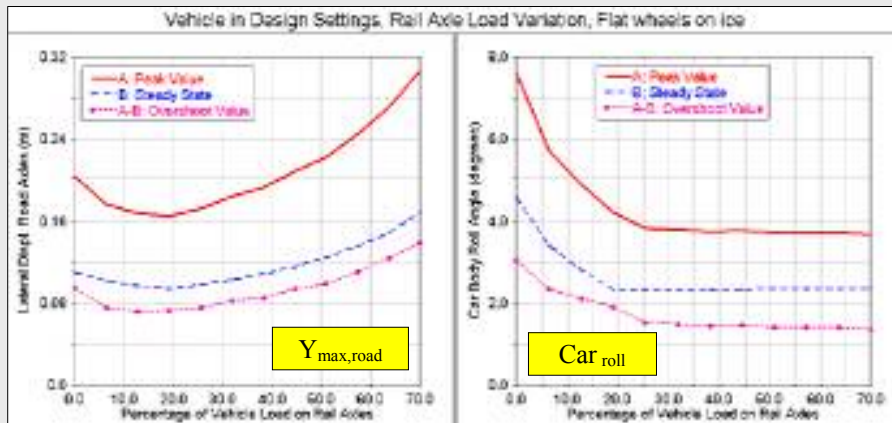


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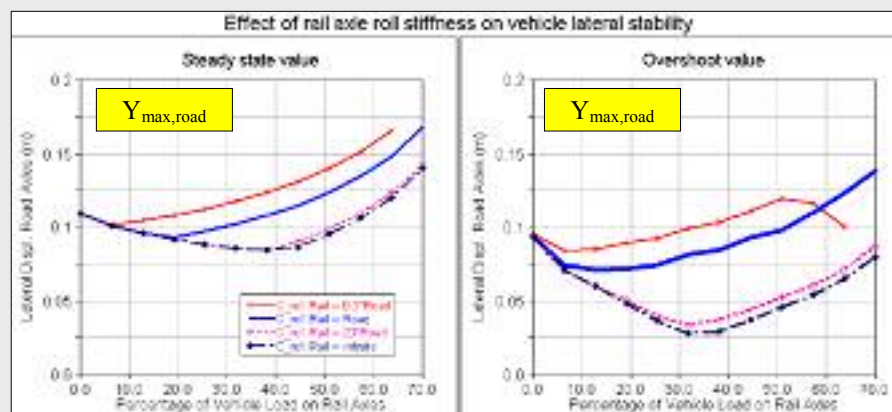
Results: 1 Reference



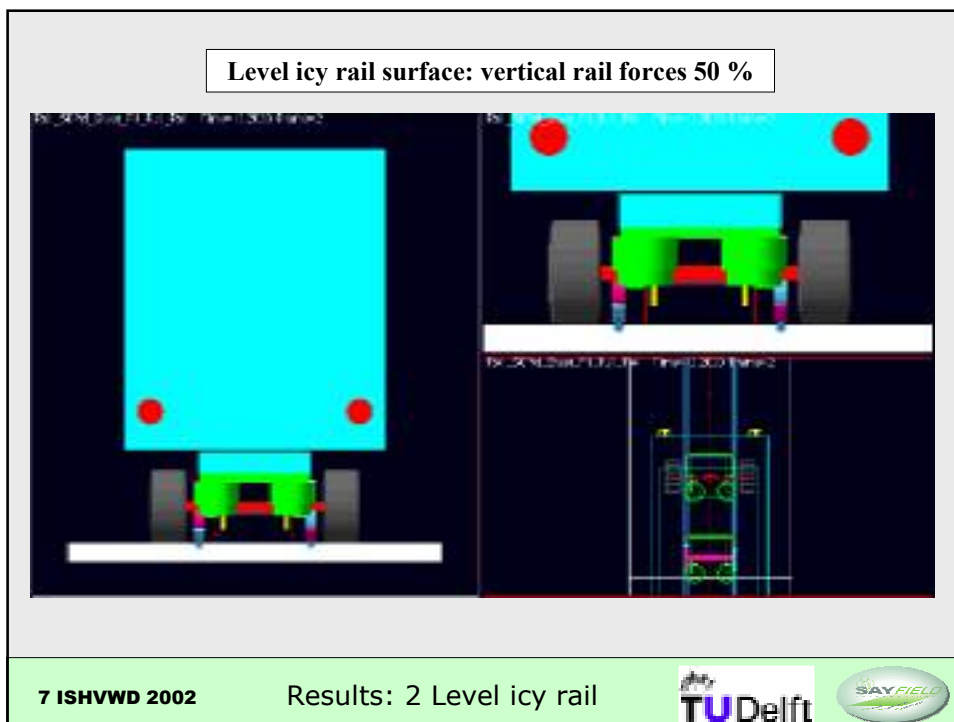
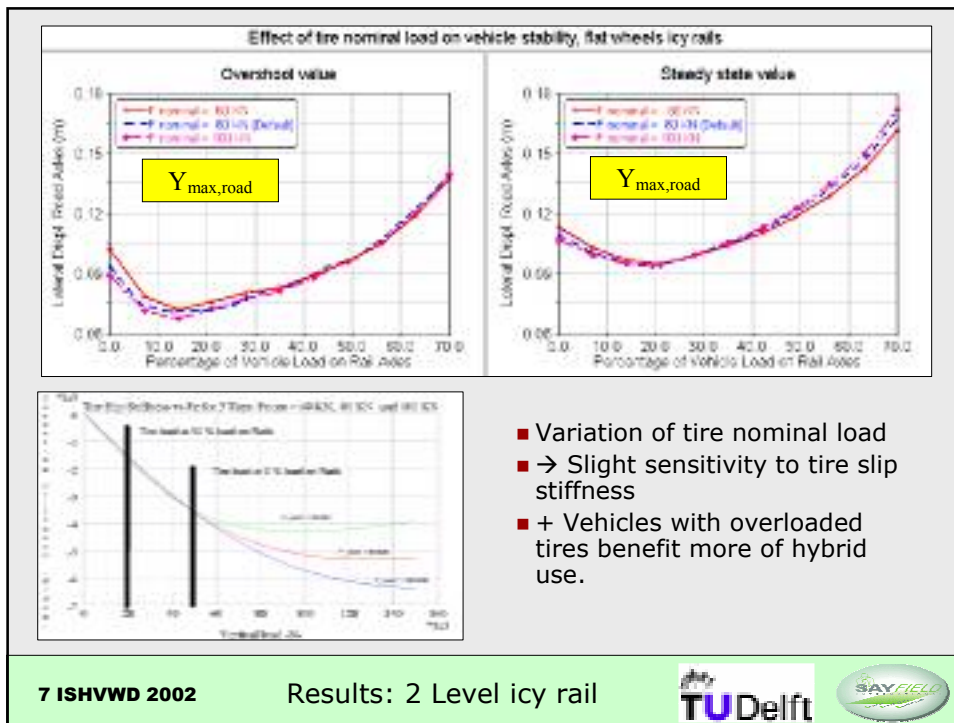
2. Level icy rail: design study results



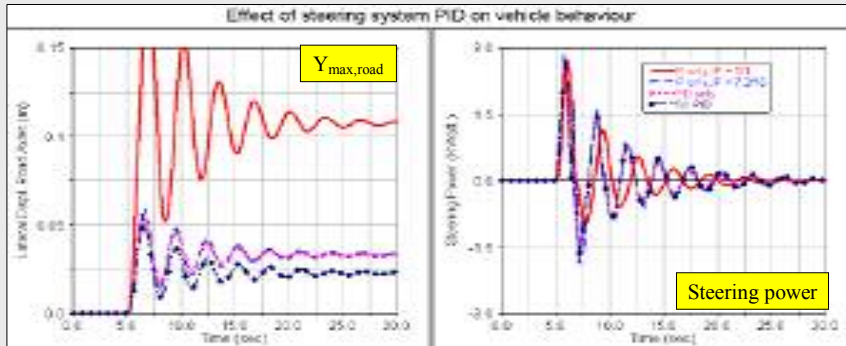
- $Y_{\max,road}$ steady state & overshoot show a minimum for $F_{z,rail,\%}$
- Transfer from road to rail axles increases C_{roll} (at $C_{roll \text{ rail}} = \text{road}$).



- High body roll stiffness improves vehicle stability
- Rail axle C_{roll} must be at least equal to road axle C_{roll} .



3. Steering parameter design study



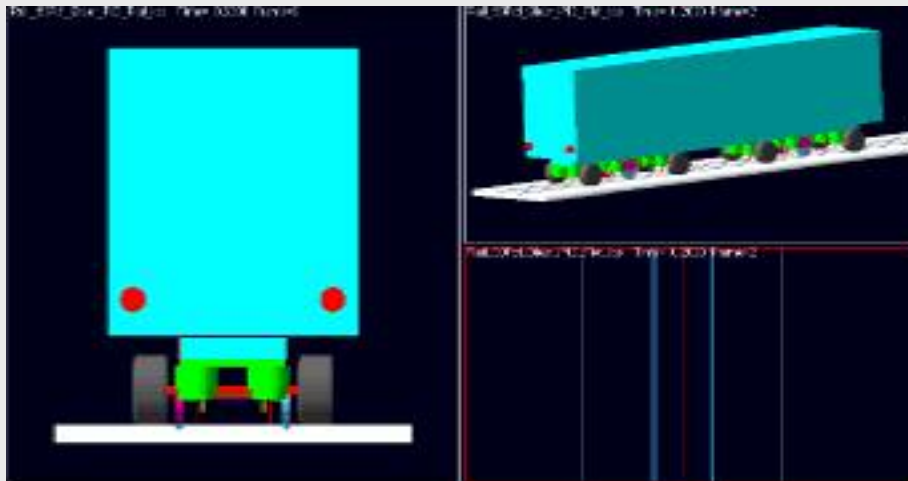
1. Red : Y deviation point follower to input angle axle 1/6: P=1
 2. Blue : P-factor axle 1/6 = 7.2/16 : strong improvement
 3. Cyan: 2 +D-factor axle 1/6 = 2.7/4.0 : slightly better
 4. Black: 3 + I-factor axle 1/6 = 8.0/5.0 : better static offset
- Factor 3-4 improvement of $Y_{max,road}$ with no extra steering power.

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Results: 3 steering PID



Hybrid with full steering PID, vertical rail forces 50 %



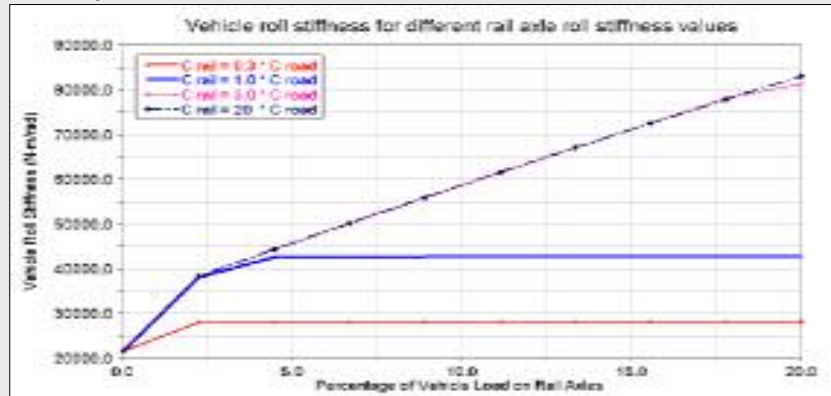
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Results: 3 steering PID



4. Complete system verification

System roll stiffness as a function of Rail %

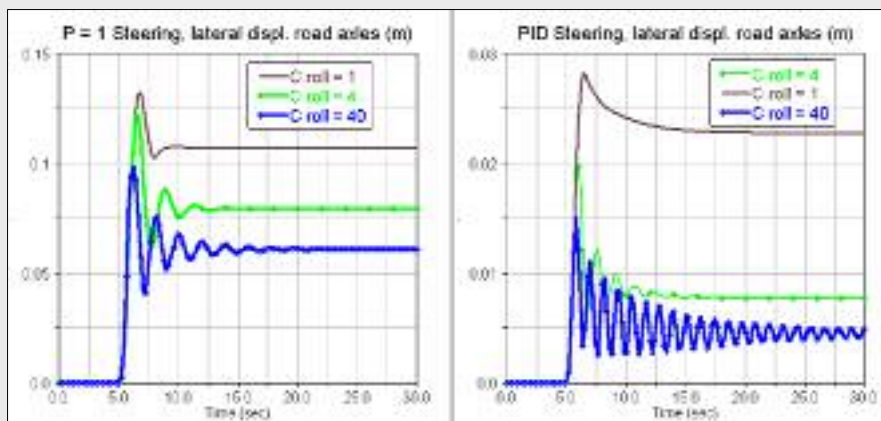


- Roll stiffness doubles at default setting: $C_{rail} = C_{road}$ axles
- At $C_{rail} \gg C_{road}$: asymptotic behavior at high % rail load.

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Effect of car roll stiffness to $Y_{max,road}$



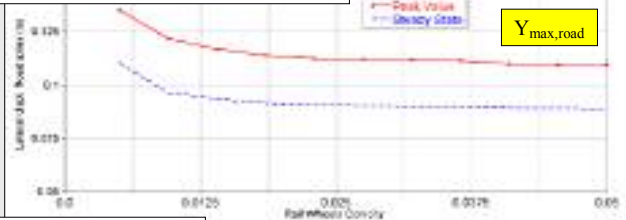
- High roll stiffness reduces $Y_{max,road}$ for both P=1 and PID.

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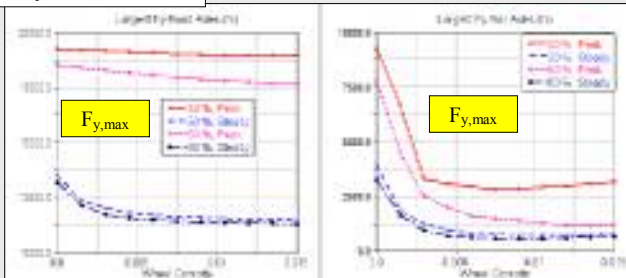
Effect of rail parameters

full rail vehicle: low F_y wind



→ Conicity 0.01 to 0.04 is OK

Hybrid vehicle



→ Conicity around 0.01 is OK

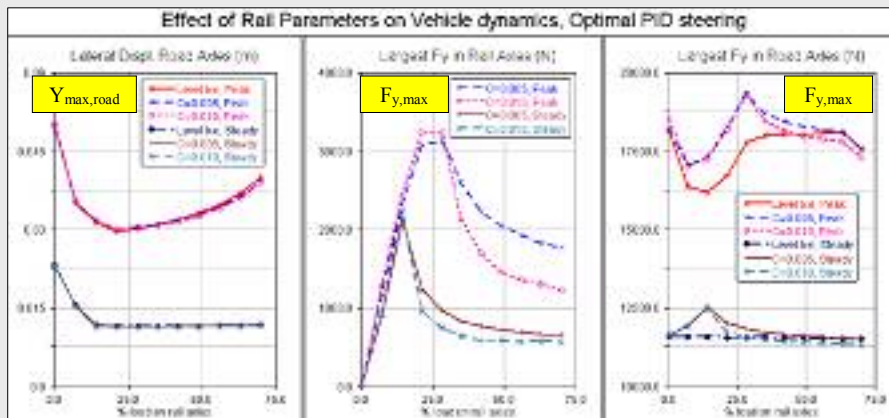
→ High % rail, low peak forces.

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Gain from Rail F_y : Level Ice / conicity = 0.005 / conicity = 0.01

Effect of Rail Parameters on Vehicle dynamics, Optimal PID steering

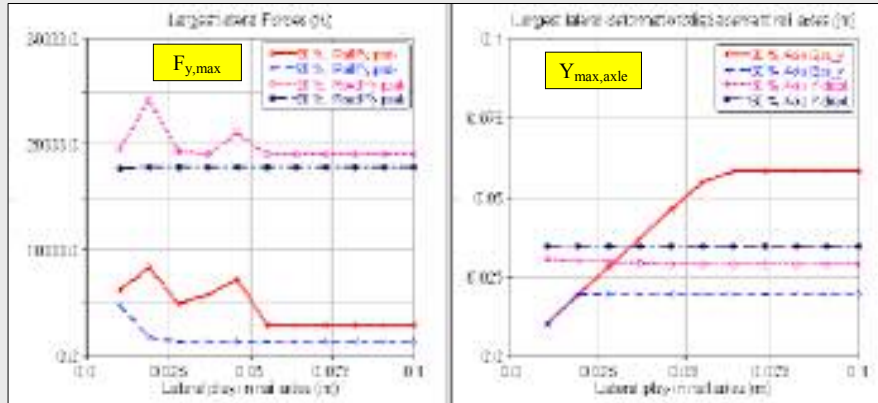


- With PID control hybrid improves at increase of rail load % !!
- Small differences due to wheel conicity & rail friction
- No extra stabilisation from rail axles required
- Light rail axles with zero lateral forces can be achieved.

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Variation of bump stop engagement for rail axle lat. play

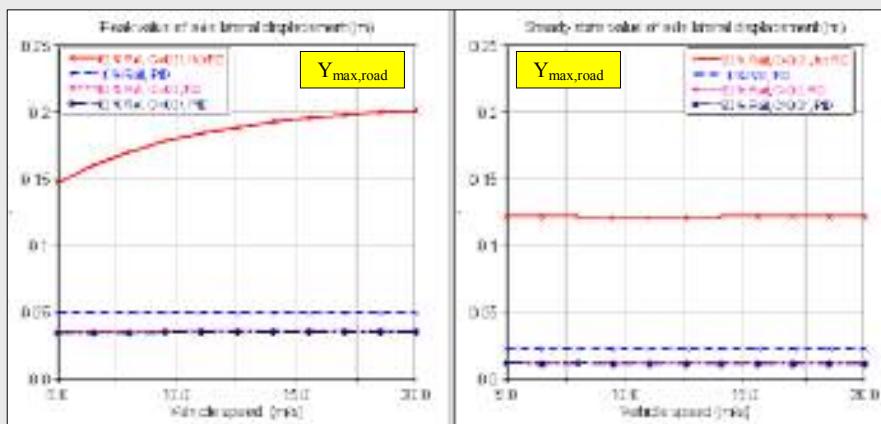


- 30 % rail: > 0.06 m play, 60 % rail: > 0.02 m play
- Self centering effect of rail helps at higher % rails
- 0.075 m play is sufficient for most circumstances.

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Final variation of velocity in various system settings



- Hybrid design is stable & predictable in desired speed range
- Speed dependency disappears due to PID steering control
- With PID control, 60 % hybrid is better than 0 % hybrid.

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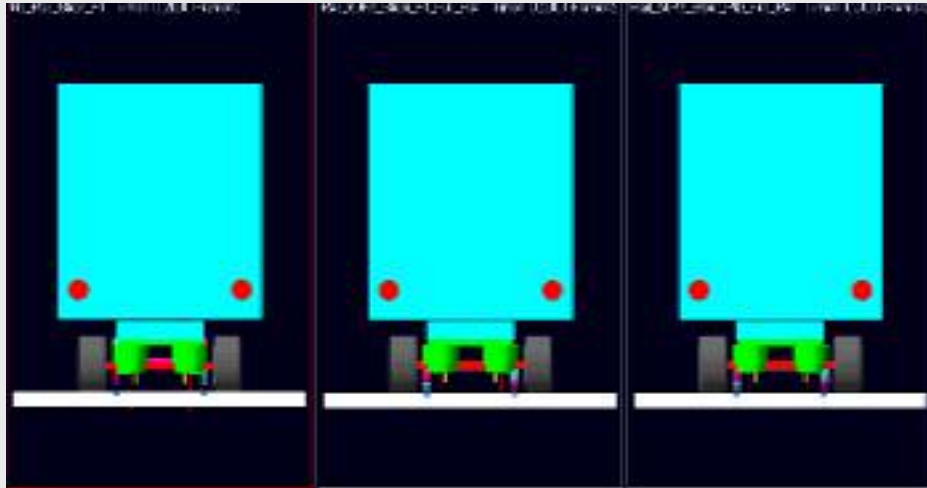


Side wind input 0.1 G: comparing three settings

P=1, no rails

P=1, 50 % rails

PID 50 % rails



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Results: 4 complete system



Conclusions & further work

- The hybrid combines the benefits of road & rail vehicles
- No dynamic problems were found in operation speed range
- W.R.T. vehicle component design:
 - Improved response is dominated by roll stiffness increase
 - System is robust w.r.t. tire mismatch problems
 - Simple P.I.D. control gives significant improvement
 - Extra stabilisation from rail guidance is small, thus minimisation of rail axle weight is feasible.
- TU Delft is now looking for partners to build a physical prototype for further tests and feasibility analysis.

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Conclusions

